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THE UNITED STATES PATENT AND TRADEMARK OFFICE

Applicant: Simon Blair Dobson

Serial No.: 09/910,475

Filed: July 20, 2001

Priority: GB 0017823.6 Filed: July 21, 2000

Group Art Unit: 3634

Examiner: Unknown

Title: CLOSURE SYSTEM

TRANSMITTAL OF CERTIFIED COPY

Assistant Commissioner for Patents
Washington, D.C. 20231

Dear Sir:

With regard to the above-referenced patent application, enclosed is a Certified Copy of prior corresponding document GB 0017823.6 filed July 21, 2000.

Respectfully submitted,

CARLSON, GASKEY & OLDS, P.C.

Karin H. Butchko

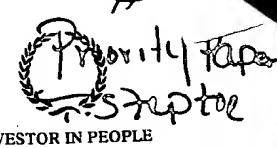
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Dated: September 20, 2001 (248) 988-8360

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Raimi Blackerby



The Patent Office
Concept House
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NP10 8QQ

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I also certify that by virtue of an assignment registered under the Patents Act 1977, the application is now proceeding in the name as substituted.

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Signed

Dated 26 July 2001



INVESTOR IN PEOPLE

GB0017823.6

By virtue of a direction given under Section 30 of the Patents Act 1977, the application is proceeding in the name of

MERITOR LIGHT VEHICLE SYSTEMS - FRANCE
Incorporated in France
105 Route d'Orleans BP48
Sully-sur-Loire 45600
France

[ADP No. 07992480001]

Request for grant of a patent

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21 JUL 00 E554589-2 D01032
P01/7700 0.00-0017823.6

The Patent Office
Cardiff Road
Newport
South Wales
NP10 8QQ

1. Your reference JBJ/P300284GB

2. Patent application number
(The Patent Office will fill in this part)

0017823.6

3. Full name, address and postcode of the or of
each applicant (*underline all surnames*)

Meritor Light Vehicle Systems (UK) Limited
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United Kingdom

Patents ADP number (*if you know it*)

APPLICATION FILED 03.07.01
7342876001

If the applicant is a corporate body, give the
country/state of its incorporation

United Kingdom

4. Title of the invention

Closure System

5. Name of your agent (*if you have one*)

"Address for service" in the United Kingdom
to which all correspondence should be sent
(*including the postcode*)

WITHERS & ROGERS
Goldings House
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London
SE1 2HW

Patents ADP number (*if you know it*)

1776001

6. If you are declaring priority from one or more
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and the date of filing of the or each of these
earlier applications and (*if you know it*) the or
each application number

Country

Priority application number
(*if you know it*)

Date of filing
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7. If this application is divided or otherwise
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application

Number of earlier application

Date of filing
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8. Is a statement of inventorship and of right to
grant of a patent required in support of this
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- a) any applicant named in part 3 is not an inventor, or
 - b) there is an inventor who is not named as an applicant,
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Priority documents

Translations of priority documents

Statement of inventorship and right to grant of a patent (Patents Form 7/77)

Request for preliminary examination and search (Patents Form 9/77)

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Any other documents
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One

11.

I/We request the grant of a patent on the basis of this application.

Signature W & R

Date 19/7/2000

12. Name and daytime telephone number of person to contact in the United Kingdom

John B Jones

0121 245 3900

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Closure System

The present invention relates to a closure systems and in particular to closure systems associated with vehicles such as land vehicles, aircraft and marine vehicles. One specific use of the invention would be in automobiles.

Automobiles are known whereby doors of such automobiles include windows which can be power opened and power closed. When a window is being power closed, it is possible for parts of people or children to be get trapped in between the window and window surround. In order for such windows to function within legal and other requirements, it is necessary to avoid applying more than a certain, predetermined maximum force to an arm, finger or other part of the body when that part of the body interferes with the normal closure of the window. Equally, it is necessary to ensure that a sufficient force is available to fully close the window into its seals. To this end, the relevant specifications allow for the limited force to be exceed once the opening is smaller than would admit entry of any part of the body.

Thus in known contact type anti-trap (anti-squeeze) systems a switch, typically within the door seal, is closed when a body part is trapped, and this triggers a decision within a control unit to stop or reverse the closing motion, and thereby release the body part.

Other known contact type anti-squeeze systems rely on an inference trap force from a parameter measurable at the motor such as change in motor speeds, change in motor current or change in output torque, all of which are related to the forces applied to the closure ie the output force of the actuator assembly.

However, due to rough roads or other terrain over which the vehicle may be travelling, primarily vertical accelerations are imposed on the vehicle. These vertical accelerations are applied to the window glass as to any other part of the vehicle and are reacted through the window regulator as variations in closure force. Under some circumstances these variations may produce forces on the window glass which resemble

the forces produced when body parts are trapped. Under such circumstances the control unit will incorrectly reopen the window then this is unnecessary. Alternatively when a body part is being trapped, accelerations on the window glass due to rough terrain may reduce the apparent trapping force to below the predetermined level where upon closure will continue and further trap a body part.

Thus situations arise which are inconvenient and/or distracting and are potentially dangerous for the occupants of the vehicle.

An object of the present invention is to provide an improved form of closure system.

Thus according to the present invention there is provided a closure system including a closure moveable for substantially closing an aperture in use, and an actuator for at least closing the closure, the actuator being mounted by mounting means, the mounting means including one or more measurements cells for measuring, in use, parameters of the closure system, in use the closure system being subjected to accelerations and being arranged such that it is possible to at least partially distinguish forces applied to the closure by the actuator from acceleration forces applied to the closure as a result of the accelerations of the closure system by consideration of the measured parameters.

Advantageously this provides for a system wherein mounting means of an actuator are able to fulfil a triple function, namely mounting of the actuator and also measurement of forces applied to the closure by the actuator and also measurement of forces applied to the closure as a result of the accelerations of the closure system.

Furthermore the applicant is the first to realise that the closure and the actuator are subjected to substantially the same accelerations and thus the actuator can be used for determining the acceleration forces on the closure. Once the acceleration forces on the closure have been determined then it is possible to subtract these from output force of the actuator to more accurately determine true trap forces.

According to a further aspect of the present invention there is provided an aperture motor assembly for at least closing an aperture, the motor assembly including measurement cells being arranged such that in use it is possible to at least partially distinguish forces applied to the associated aperture closure by the actuator from accelerations forces applied to the associated aperture closure as a result of accelerations of the aperture closure and motor assembly by consideration of the output from the measurement cells.

The invention will now be described by way of example only, with reference the accompanying drawings in which:-

Figure 1 is a side view of a vehicle including closure system according to the present invention;

Figure 2 is a cross section view of the window motor and gearbox of figure 1 and

Figure 3 is a further view of figure 2.

With reference to figure 1 there is shown a vehicle 10 having a door 12 with a window aperture 14. An aperture closure in the form of a window glass 16 is moveable vertically to open and close the window aperture 14. A window regulator shown generally as 18 includes a window motor 20 and a gearbox 22.

The motor 20 and gearbox 22 are mounted via mounting means, in this case in the form of first mount 24 and second mount 26. First and second mounts 24 and 26 include load cells in this case shear load cells C1 and C2 each forming the load reaction path between the motor and the vehicle door, and between them constrain the motor within the door.

The shear load cells C1 and C2 are positioned at distance $2R$ from each other and the geometrical position of the centre-of-gravity CG of the motor/gearbox is also known.

Consideration of figure 2 shows that the shear load cells C1 and C2 have output levels S1 and S2.

An output torque T from the motor acts clockwise around the motor output shaft when the window is being closed and, with the vehicle 10 stationary, is a function of a output force f.

The weight of the gearbox and window motor acts through the centre-of-gravity CG. With the vehicle 10 stationary the force M at CG is equivalent to the combined weight of the motor 20 and gearbox 22. However, with the vehicle moving over rough terrain the force M will vary. It should be noted that CG is located horizontally by distance x from shear load cell C1 and vertically from shear load cell C1 by distance z.

Consideration of figure 2 shows that the forces S1 and S2 can be resolved in the x and y directions and become S_{1x}, S_{1z}, S_{2x} and S_{2z}. The output torque T of the motor can be considered to be a tangential force F_t acting at a radius r equivalent to the pitch circle diameter of a drum or pinion and, for convenience, this has been shown to be parallel to the z axis.

Note that the analysis below is more complicated where the sensors and output shaft are not in line, and/or the forces act at some other angle, but it may demonstrated that the same principles apply.

Consideration of the above shows that several equations can be written, which express the situation at steady state ie with the vehicle 10 stationary and the window closing at a constant speed.

Given that we do not know the directions of S1 and S2 merely their magnitudes then, by pythagoras:-

$$S_{1x}^2 + S_{1z}^2 = S_1^2 \text{ and } S_{2x}^2 + S_{2z}^2 = S_2^2$$

Since we have conveniently defined F_t parallel to z axis and M acting vertically downwards, also parallel to the z axis then $F_t x = 0$ and $F_t z = F_t$.

Furthermore $F_{mx} = 0$ and $F_{mz} = M$.

Resolving in x and z we have the summations:

$$\Sigma x = 0, \Sigma z = 0$$

thus

$$S_1x + S_2x = 0, \quad S_1z + S_2z = F_t - M$$

And taking moments about S1

$$S_2z \cdot 2R + F_t \cdot (R+r) = M \cdot x$$

Collecting Terms

$$S_2z = F_t \cdot (R+r)/2 \cdot R - M \cdot x/2 \cdot R$$

of which R , r & x are all known and constant for a given application.

$$S_2z = F_t \cdot k_1 - M \cdot k_2$$

Where the constants $k_1 = (R+r)/2 \cdot R$ & $k_2 = x/2 \cdot R$;

Thus

$$S_1z = F_t \cdot M - S_2z$$

$$S_1z = F_t \cdot M - F_t \cdot (R+r)/2 \cdot R - M \cdot x/2 \cdot R$$

Collecting Terms

$$S_1x = F_t \cdot F_t \cdot (R+r)/2 \cdot R - M \cdot M \cdot x/2 \cdot r$$

$$S_1z = F_t \cdot (1 \cdot (R+r)/2 \cdot R) - M \cdot (M \cdot x/2 \cdot R)$$

But $k_1 = (R+r)/2 \cdot r$ & $K_2 = x/2 \cdot R$, so

$$S_1z = F_t \cdot (1 - k_1) - M \cdot (1 - k_2)$$

following which

Thus $S_2 = F_t \cdot k_1 - M \cdot k_2$

and

$$S_1 = F_t(1-k_1) - M(1-k_2)$$

S_1 and S_2 from the output from the shear load cell C_1 and C_2 , k_1 and k_2 being constant, we now have two equations and two unknowns (F_t and M) and therefore can solve for F_t and M .

This solution allows comparison of the motor/gearbox effective weight M and the known pre-measured value of the motor/gearbox weight. This comparison gives an instantaneous value for the vertical g-forces applied to the window motor and therefore the adjacent window glass and permits greater discrimination of the system loads resulting from the vehicle movement from those associated with a trapped object or body part.

Thus the above system, by comparing the output S_1 and S_2 from the shear load cell C_1 and C_2 , the proportion of the measured output due to vertical acceleration and that due to an object trapped may be distinguished arithmetically and thus a better definition of actual trap force (as opposed to apparent trap force) maybe obtained. As such it is possible to largely eliminate interference with the true trap force signal caused by vibration and/or accelerations with a large vertical component. The present invention achieves this in a particular cost effective manner as a minimum of components are required since the shear load cells C_1 and C_2 provide both the function of mounting the motor and also of measuring the parameters which can be used to determine true trap force from apparent trap force.

As a result of improved sensitivity of the system to objects being trapped, lower force thresholds may be specified and therefore more rapid reaction may be obtained both of which leads to a reduction in the overall trap force experienced by the person or object.

The system also reduces the likelihood of false trap signals and hence false reopening of the window, thus reducing the possibility of distraction and annoyance to occupants of the vehicle.

Once the window is virtually closed such that any gap between the window glass and window aperture is sufficiently small to not allow entry of a small body part such as a finger, then the anti-squeeze requirement is no longer necessary. Thus when the window glass reaches such a position this position can be indicated by a proximity sensor, micro switch or the like which would indicate to control means of the window that anti-squeeze is no longer required for the final closing of the window.

It is envisaged that the present invention could be used in a variety of applications, such as automotive windows and other partitions moving in a primarily vertical direction. However, the principles outlined above would be applicable to other types of closure where the motor may be so mounted as to be subject to the disturbing forces in the same manner as the closure being operated by that motor. Further applications include vehicular sun roofs, transverse and longitudinal sliding doors as typically used for corridor and compartment closures in trains and aircrafts, and other types of sliding partition on ships and other marine vehicles.

Claims

1. A closure system including a closure moveable for substantially closing an aperture in use, and an actuator for at least closing the closure, the actuator being mounted by mounting means, the mounting means including one or more measurement cells for measuring, in use, parameters of the closure systems, in use the closure system being subjected to accelerations and being arranged such that it is possible to at least partially distinguish forces applied to the closure by the actuator from acceleration forces applied to the closure as a result of the accelerations of the closure system by consideration of the measured parameters.
2. A closure system as defined in Claim 1 in which the closure system is part of a vehicle.
3. A closure system as defined in Claim 2 in which the vehicle is a land vehicle or a aircraft or a marine vehicle.
4. A closure system as defined in any preceding claim in which the closure moves primarily in a vertical direction.
5. A closure system as defined in any one of Claims 1 to 4 in which the closure moves primarily in a horizontal direction.
6. A closure systems as defined in any preceding claim in which the closure is a window.
7. A closure system as defined in Claim 6 in which the window is a door window.
8. A closure systems as defined in Claim 7 in which the closure system is mounted in the door.
9. A closure systems as defined in any one of Claims 1 to 5 in which the closure is a sun roof.

10. A closure system as defined in any one of Claims 1 to 5 in which the closure is a partition.
11. A closure system as defined in any one of Claim 1 to 5 in which the closure is a sliding personnel door.
12. A closure system as defined in any preceding claim in which the actuator includes an electric motor.
13. A closure system as defined in Claim 12 in which the actuator includes a gearbox.
14. A closure system as defined in Claim 13 in which the actuator is a window regulator motor including a gearbox.
15. A closure system as defined in any preceding claim in which the mounting means comprises at least two measurement cells positioned in a spaced apart relationship.
16. A closure system as defined in any preceding claim in which the parameters measured are force.
17. A closure system as defined in any preceding claim in which the actuator is capable of opening the closure.
18. A closure system as defined any preceding claim further including an aperture to be closed by the closure.
19. An aperture motor assembly for at least closing an aperture, the motor assembly including measurement cells being arranged such that in use it is possible to at least partially distinguish forces applied to the associated aperture closure from acceleration forces applied to an associated aperture closure as a result of accelerations of the aperture closure and motor assembly by consideration of the output from the measurement cells.

20. An aperture motor assembly as defined in Claim 19 suitable for use as a window regulator motor assembly.
21. An aperture assembly as defined in Claim 20 in which the motor assembly includes a gearbox.

Abstract

A closure system (8) including a closure (16) moveable for substantially closing an aperture (14) in use, and an actuator (20) for at least closing the closure, the actuator being mounted by mounting means (24, 26), the mounting means including one or more measurement cells (C1, C2) for measuring, in use, parameters of the closure systems, in use the system being subjected to movement and being arranged such that it is possible to at least partially distinguish trap forces applied to the closure from acceleration forces applied to the closure as a result of the movement of the closure system by consideration of the measured parameters (Figure 2).

